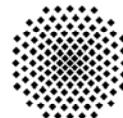

Modelling Thermal-Physiological Responses using CFD

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T Yang, P Cropper, R Yousaf, D Fiala

CIBSE Building Simulation Group
2 December 2009



Universität Stuttgart
Germany

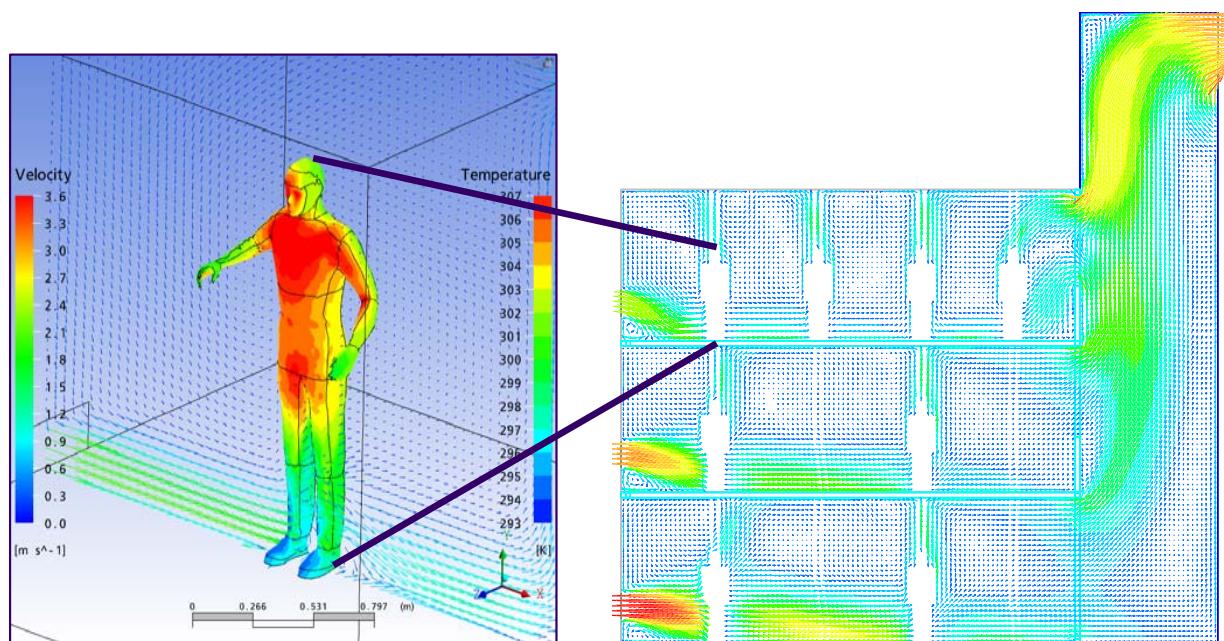


Presentation Content

- Project background
- The thermal-physiological model
- The CFD model
- Coupling technology
- Coupled simulation system demonstration
- Natural ventilation application
- Future work

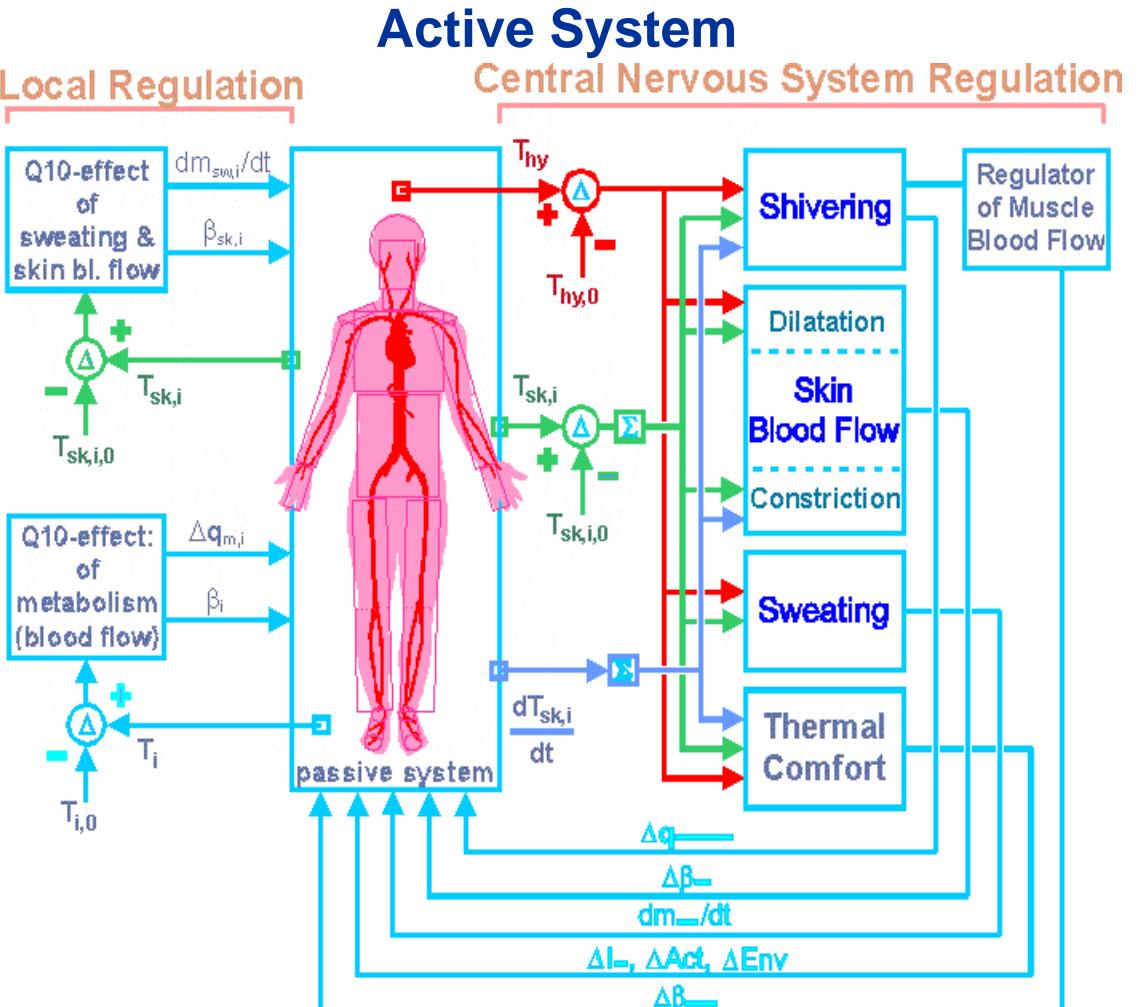
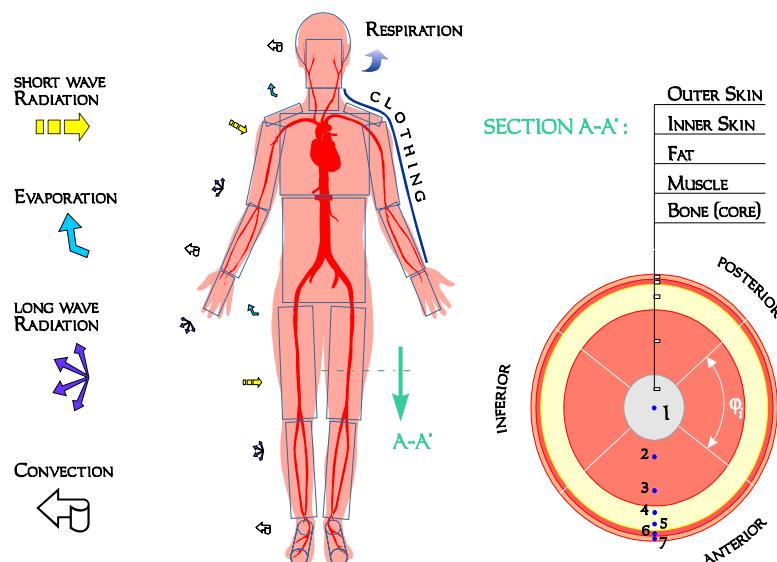
Project Background

- Aim: *to develop a validated simulation system capable of predicting impact of natural ventilation designs on occupants and vice versa*

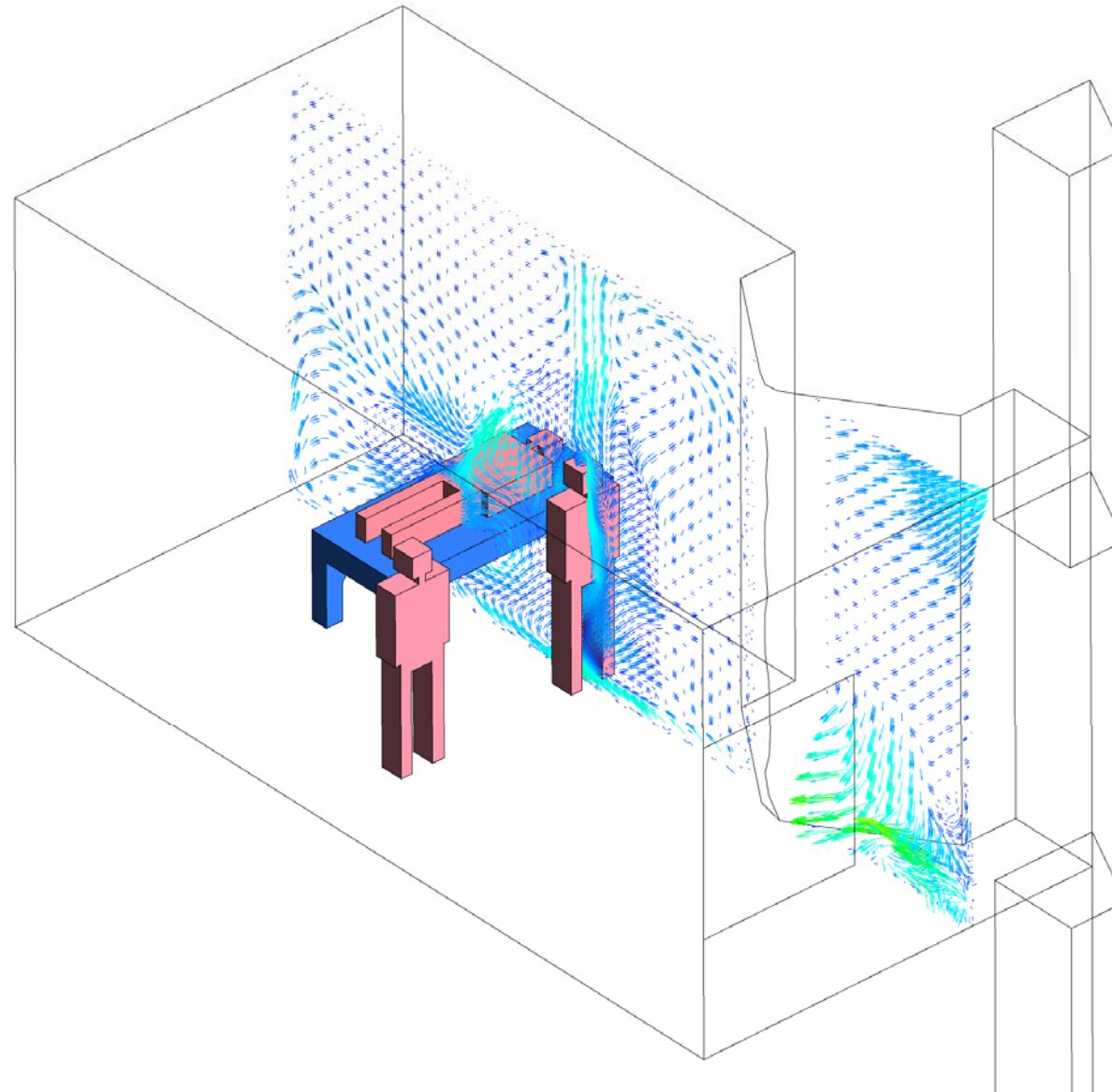


IESD-Fiala Human Thermal Comfort Model

Passive System

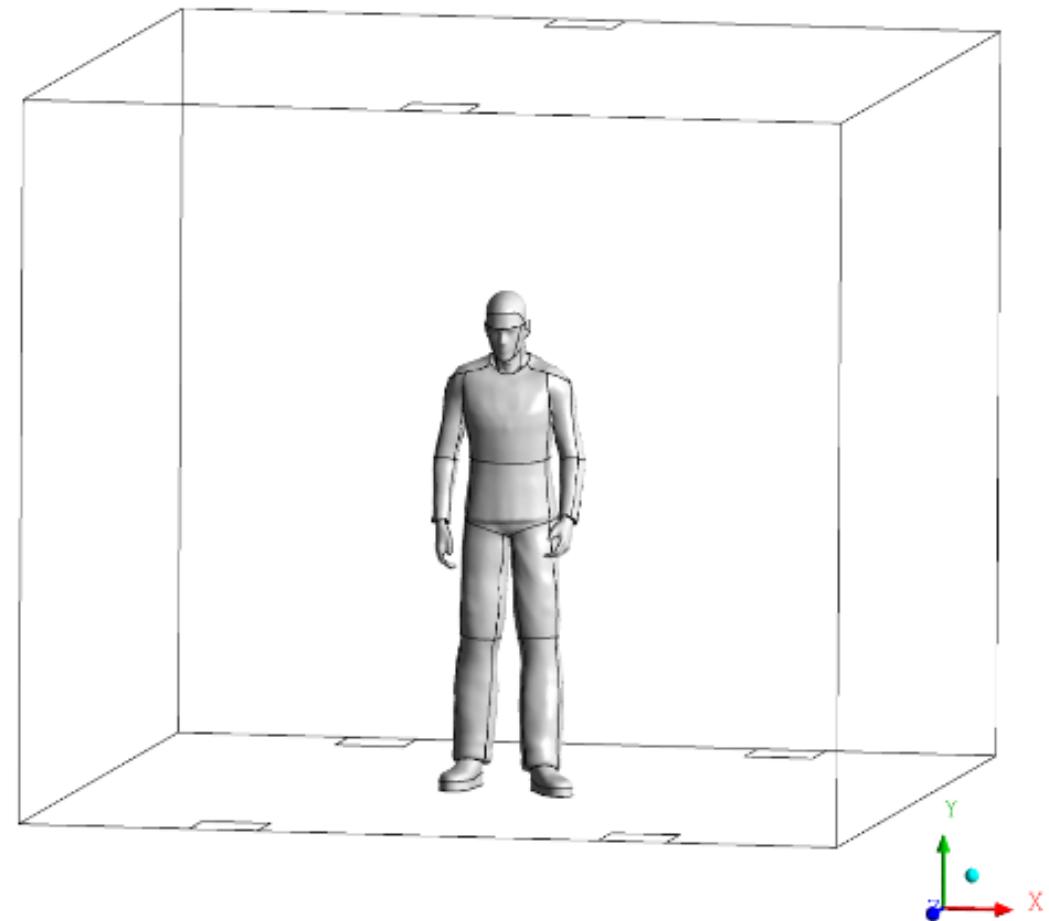
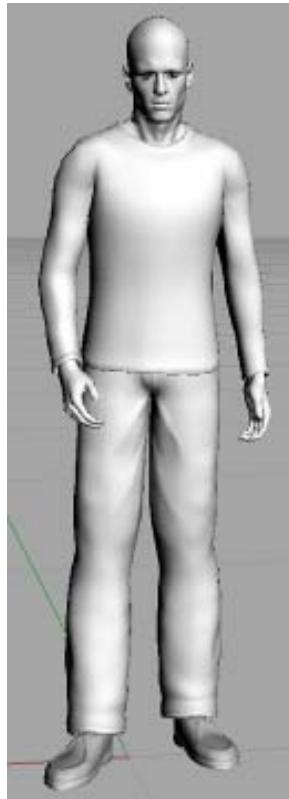


Computational Fluid Dynamics (CFD)

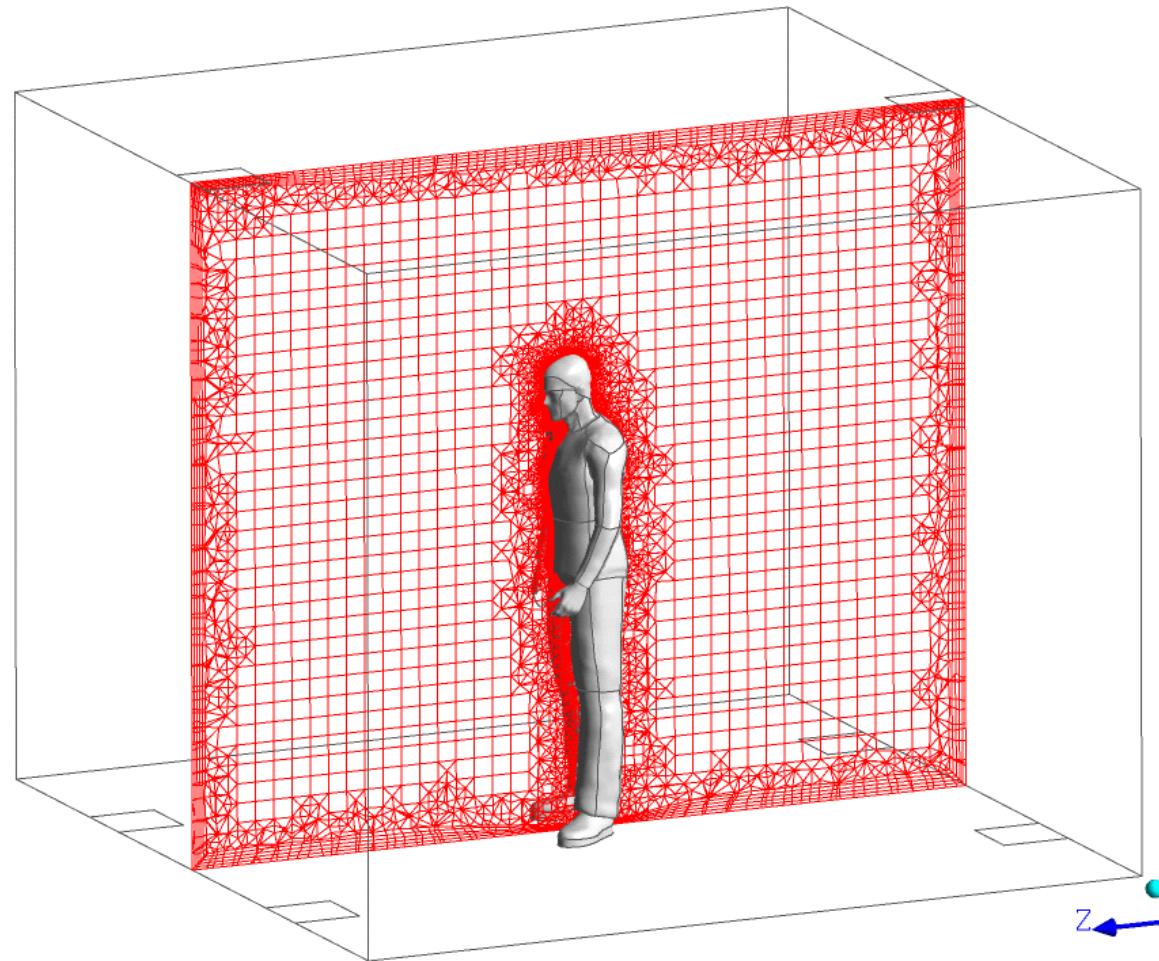
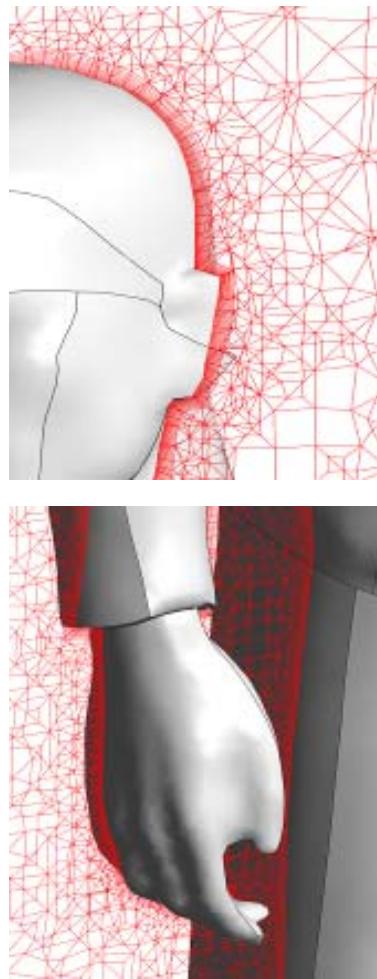


PMV
PPD
DTS

Computational Manikins



The Manikin in a CFD Environment



ANSYS CFX - Customisation

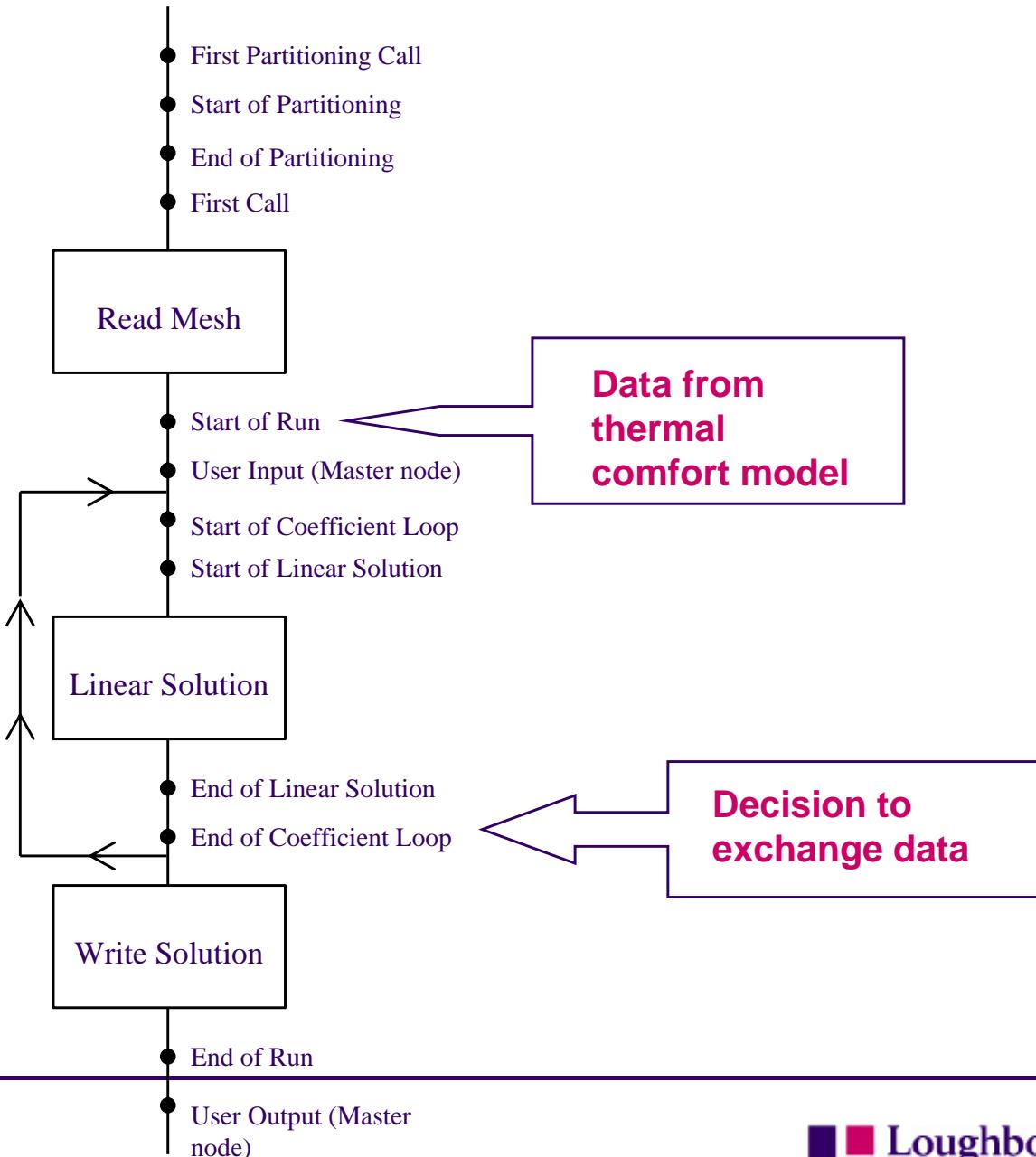
- Chosen CFD solver is ANSYS CFX
 - Powerful Application Program Interface (API)
 - Enables customisation of the solver solution cycle

- CFX can be customised using
 - CFX Command Language (CCL)
 - CFX Expression Language (CEL)
 - Embedded Perl
 - User Fortran

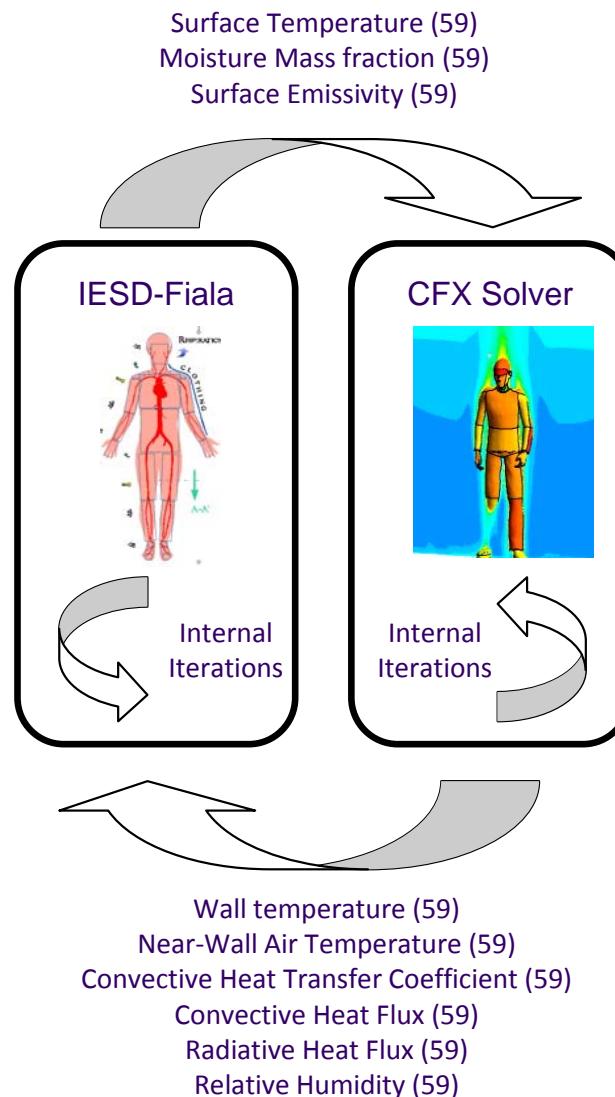
ANSYS CFX - User Fortran

- Two types of user Fortran
 - User CEL (user written CEL functions)
 - Junction Box (subroutines called at specific points in the solver execution cycle)
- Both are Fortran subroutines
- Both have access to all solver data structures
- User subroutines may, in turn call:
 - other user written subroutines
 - any solver subroutine or function

CFD Solver Events

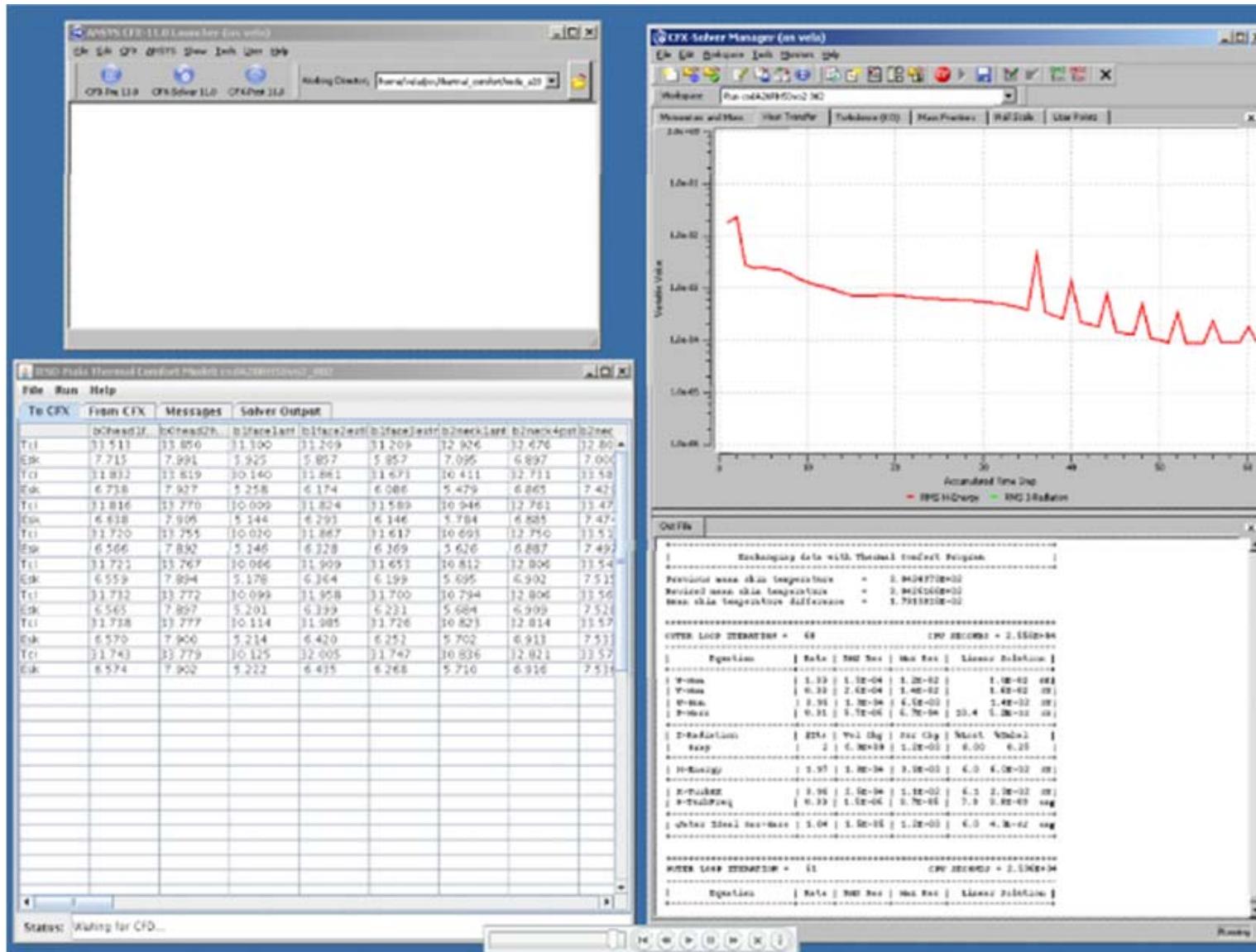


Data Exchange

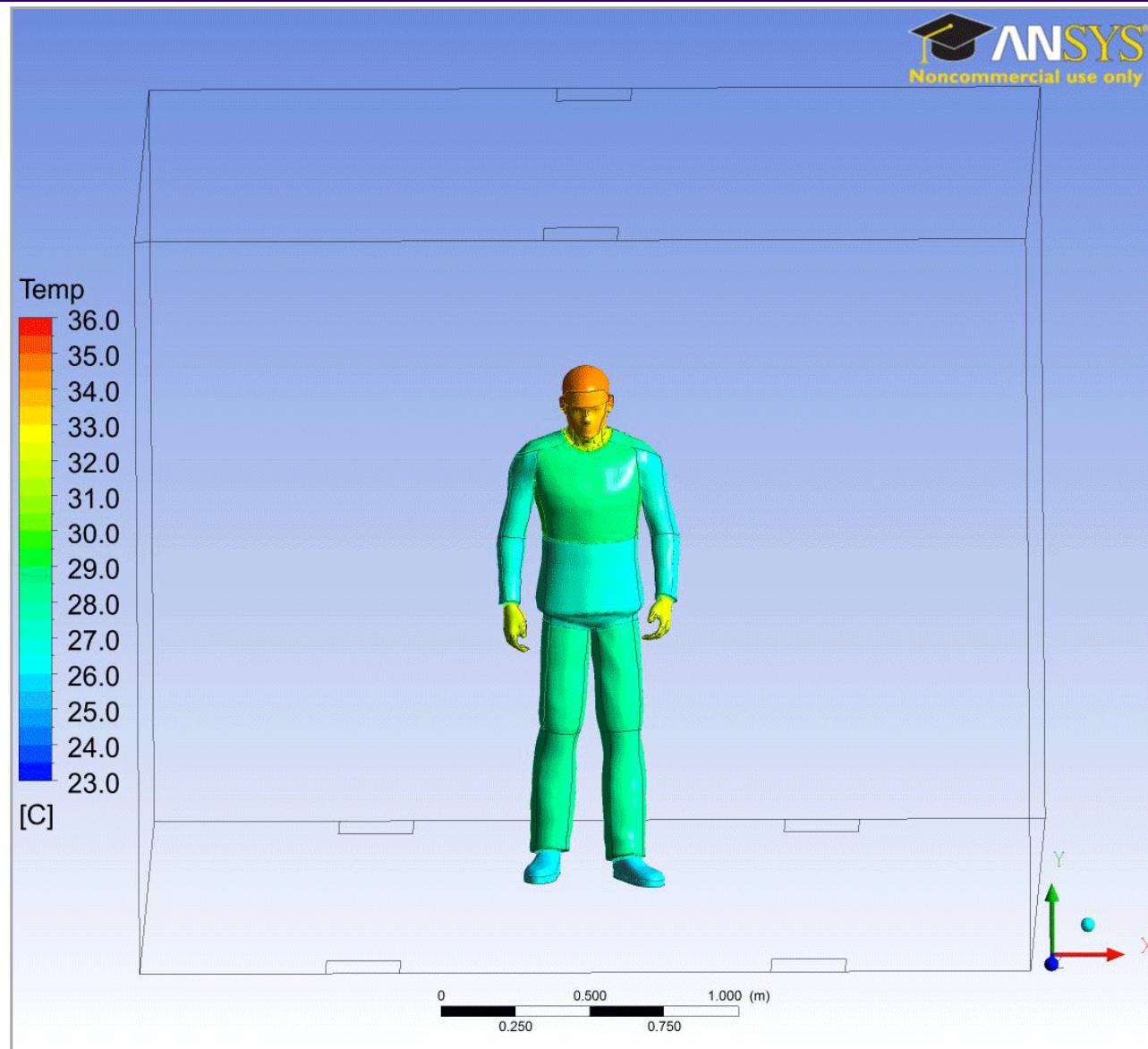


Exchange data while
10-3 > RMS > 10-4

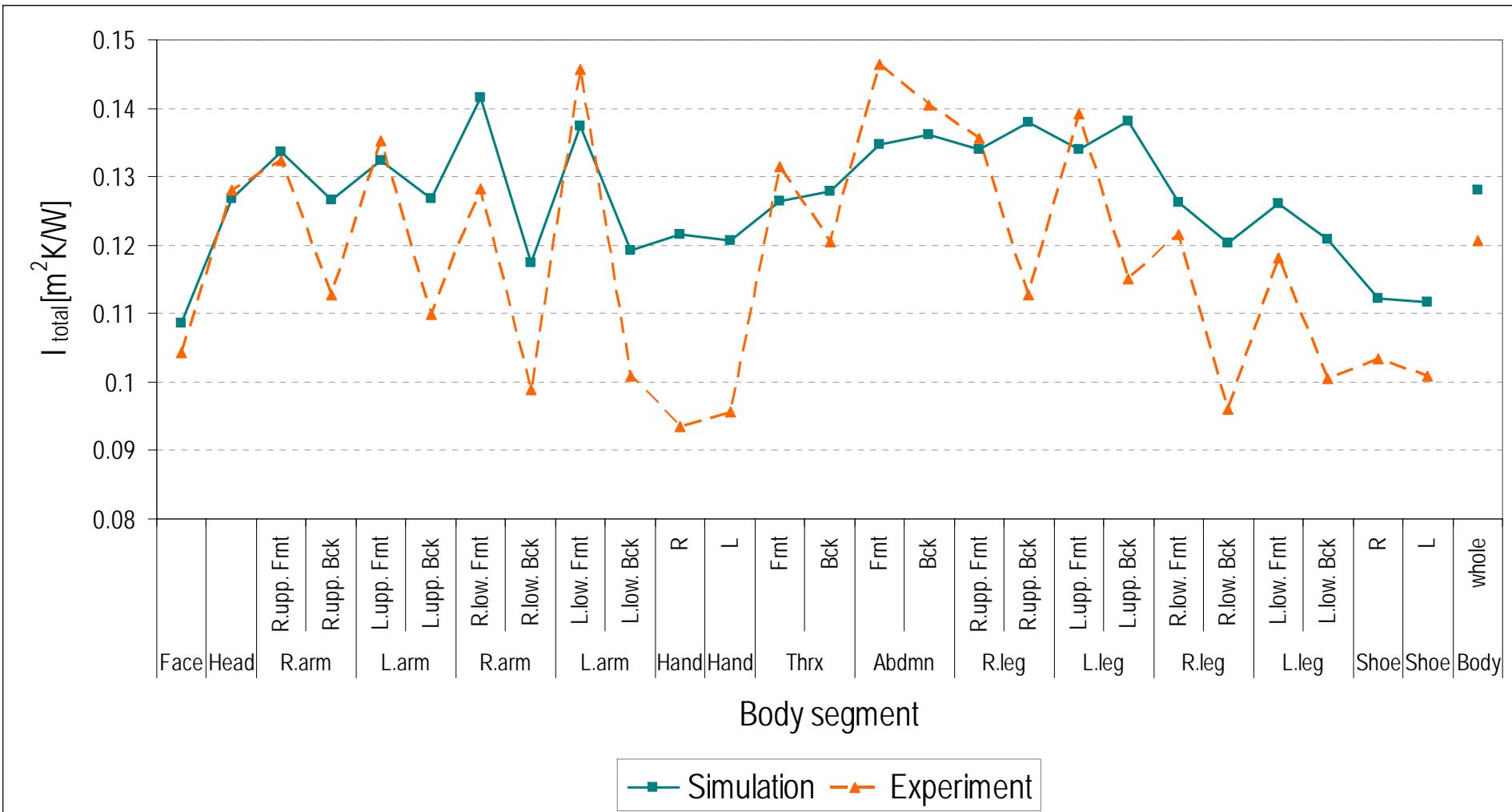
The Coupled System Demo



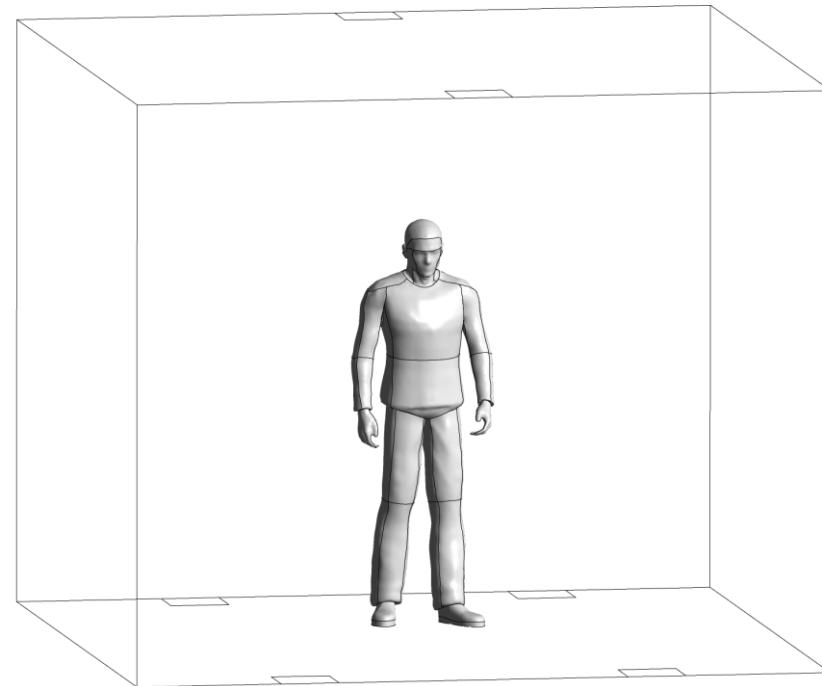
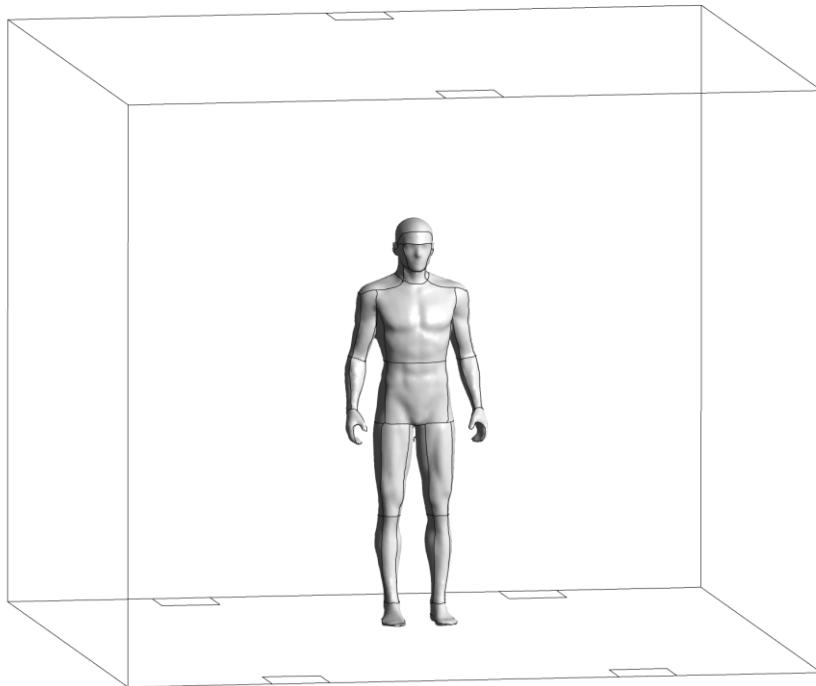
Coupled System Convergence



Validation: nude case



Natural Ventilation Case Study



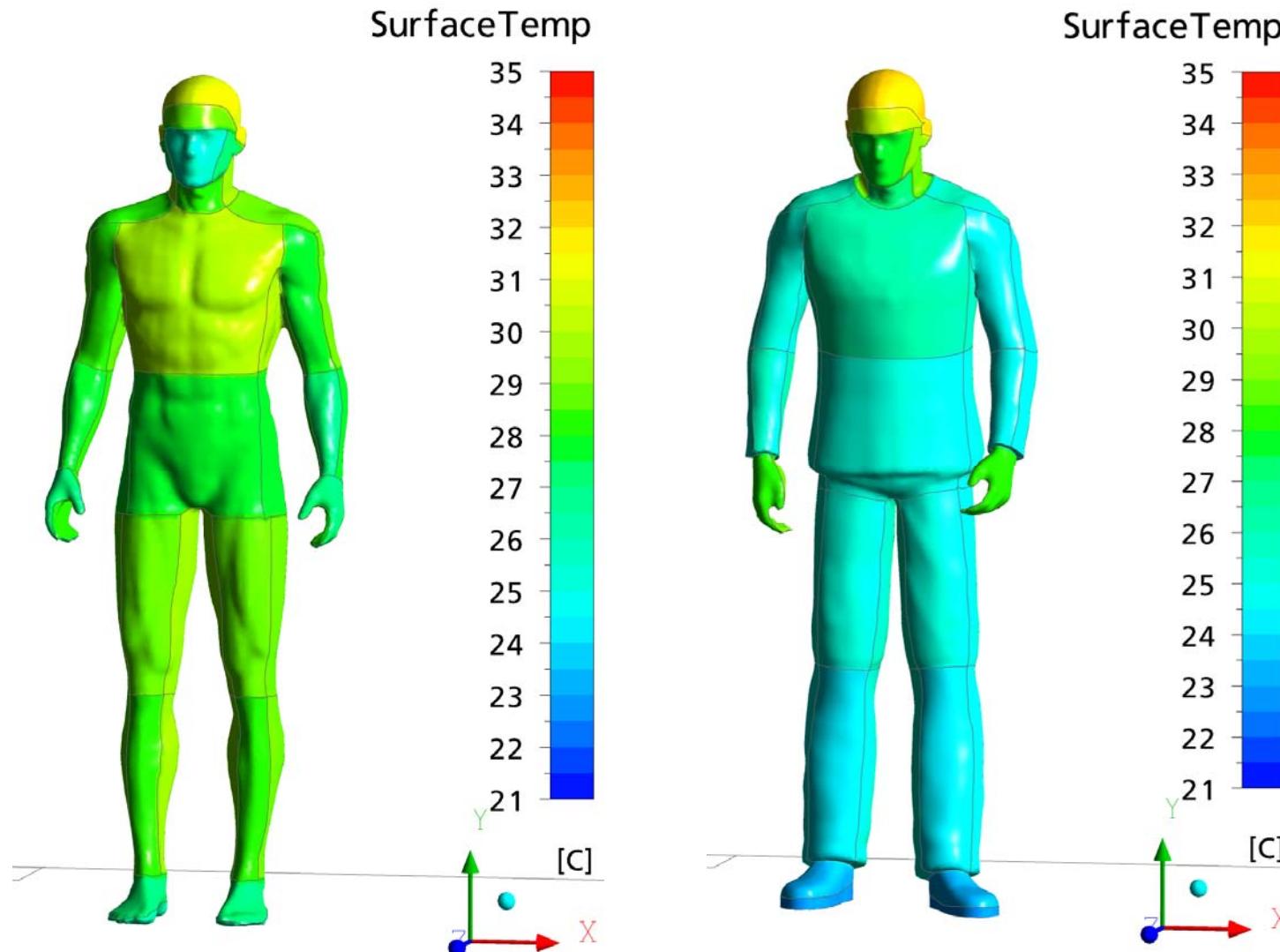
- Room (3m×3m×2.5m)
- Vent (0.25m×0.25m)
- Manikin
 - located in the centre
 - placed 0.06m above the floor

- Manikin 1.74m, 66kg
- 1.79m² (nude)
- 2.11 m² (clothed)
- $f_{cl}^* = 1 + 0.305 \times I_{cl} = 1.18$
- $I_{cl} = 0.6 \text{ clo}$

* I Holmer, H Nilsson, G Havenith, KC Parsons (1999) Clothing convective heat exchange - proposal for improved prediction in standards and models, *Annals of Occupational Hygiene* vol 43 number 5, pp 329-337

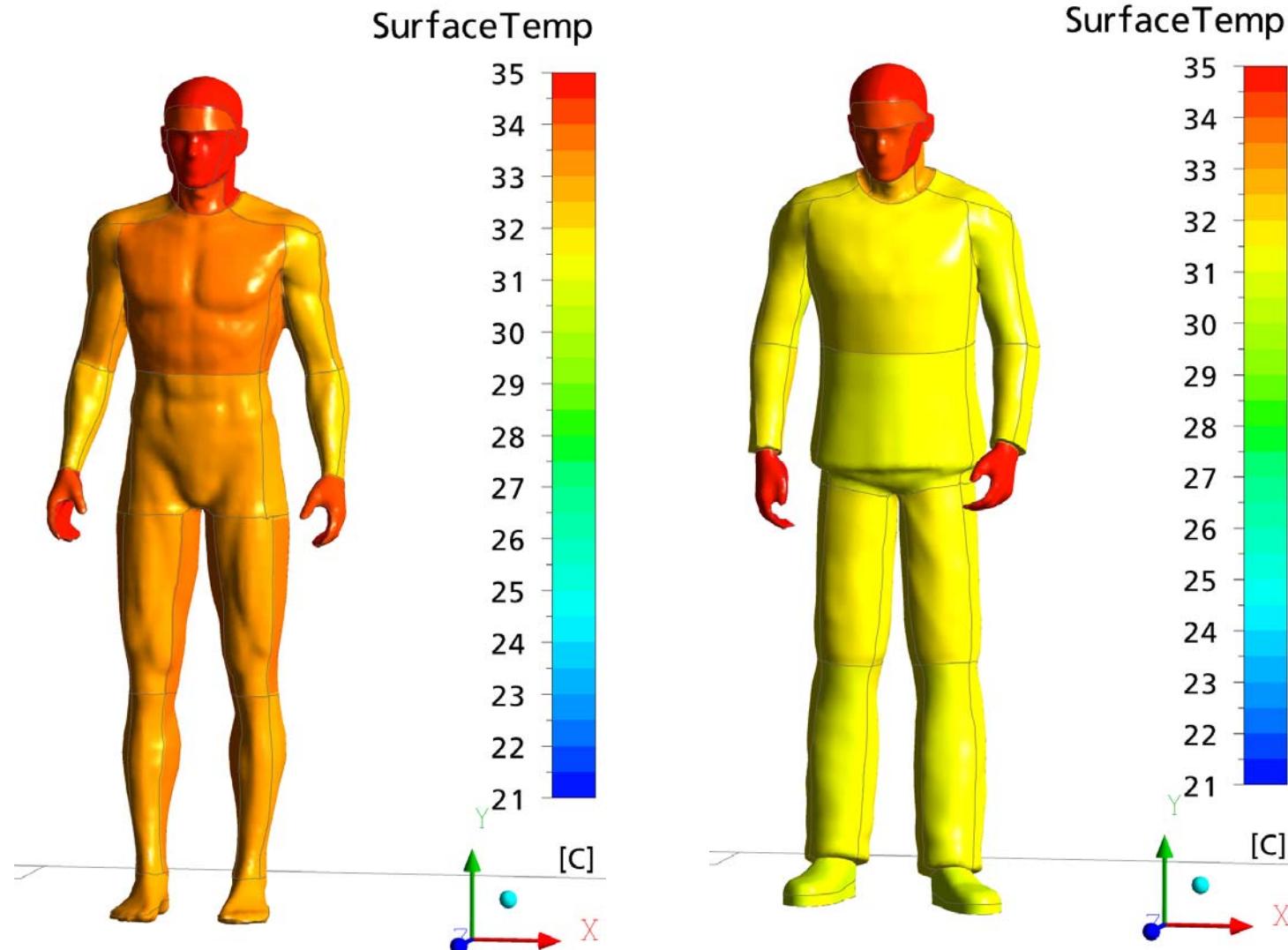
Natural Ventilation Case Study: Results

[T_{air} =21°C RH=40%]



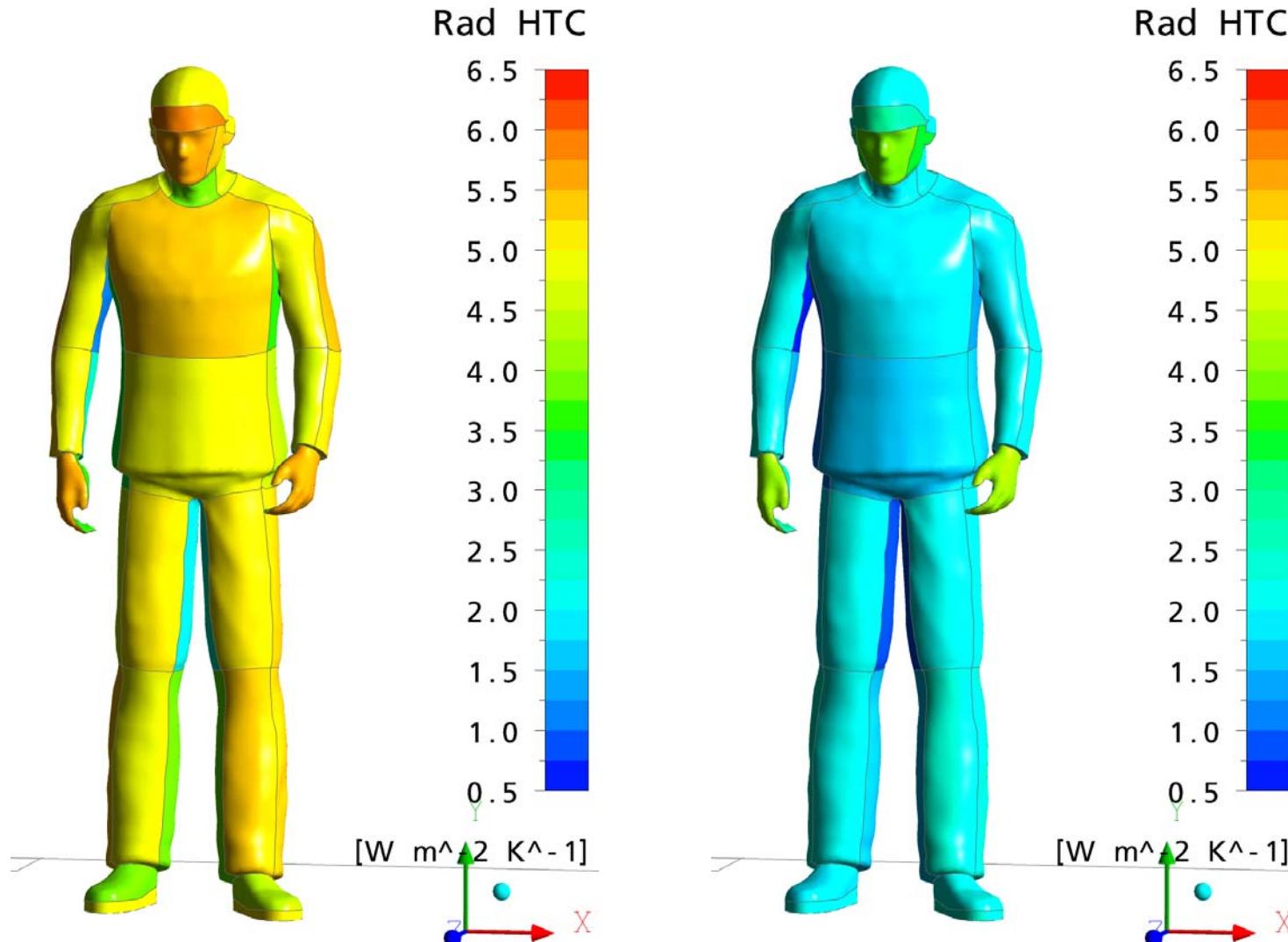
Natural Ventilation Case Study: Results

[new – Tair =30°C RH=40%]

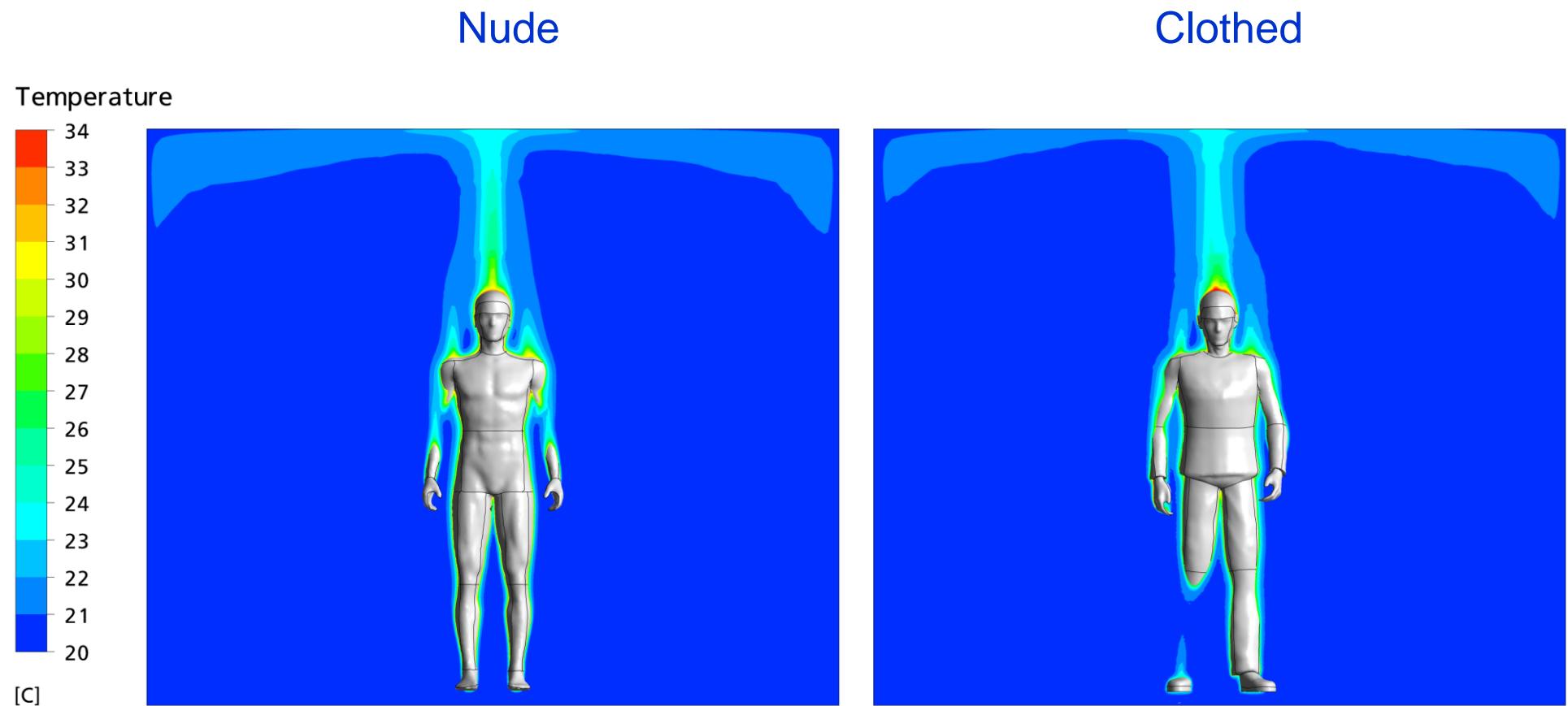


Natural Ventilation Case Study: Results

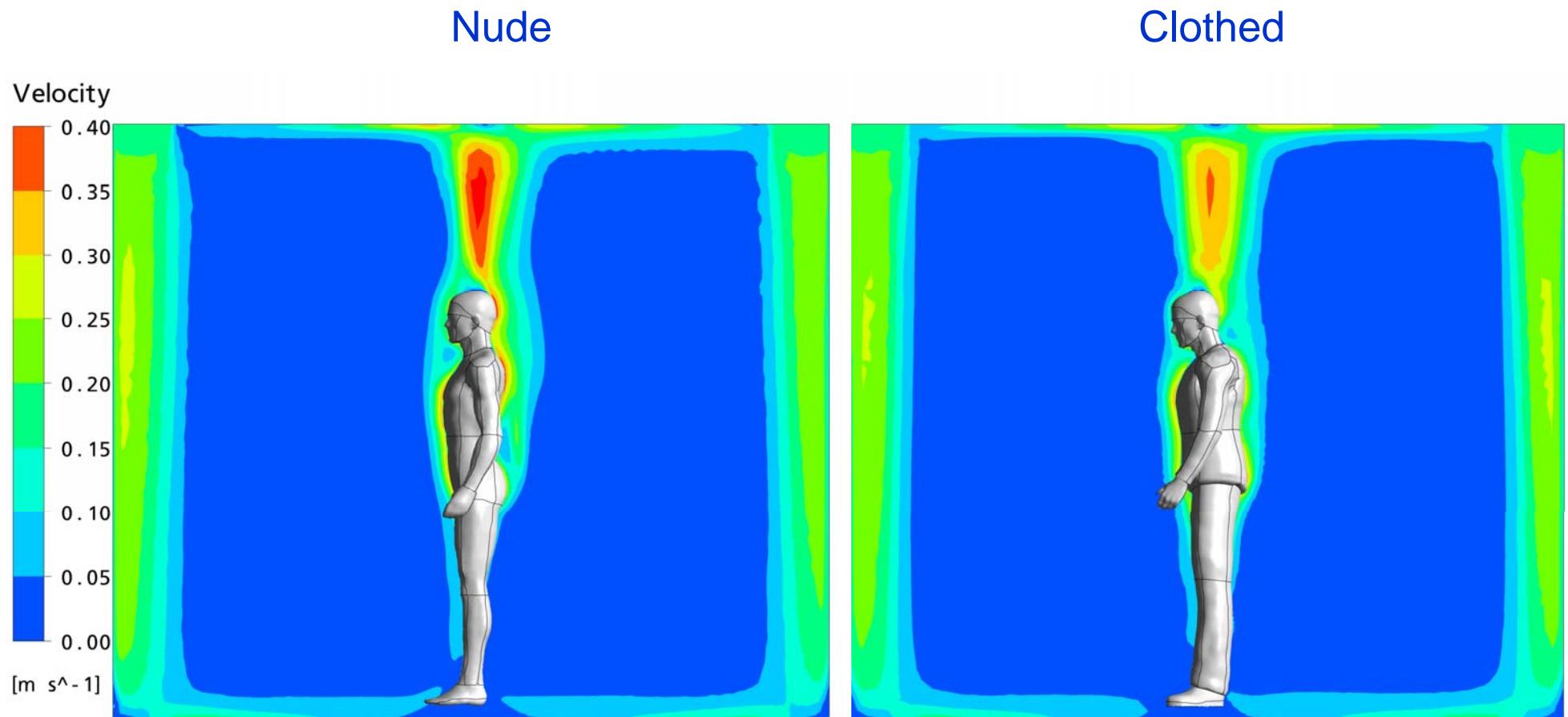
Radiative htc [T21clo vs T30clo]



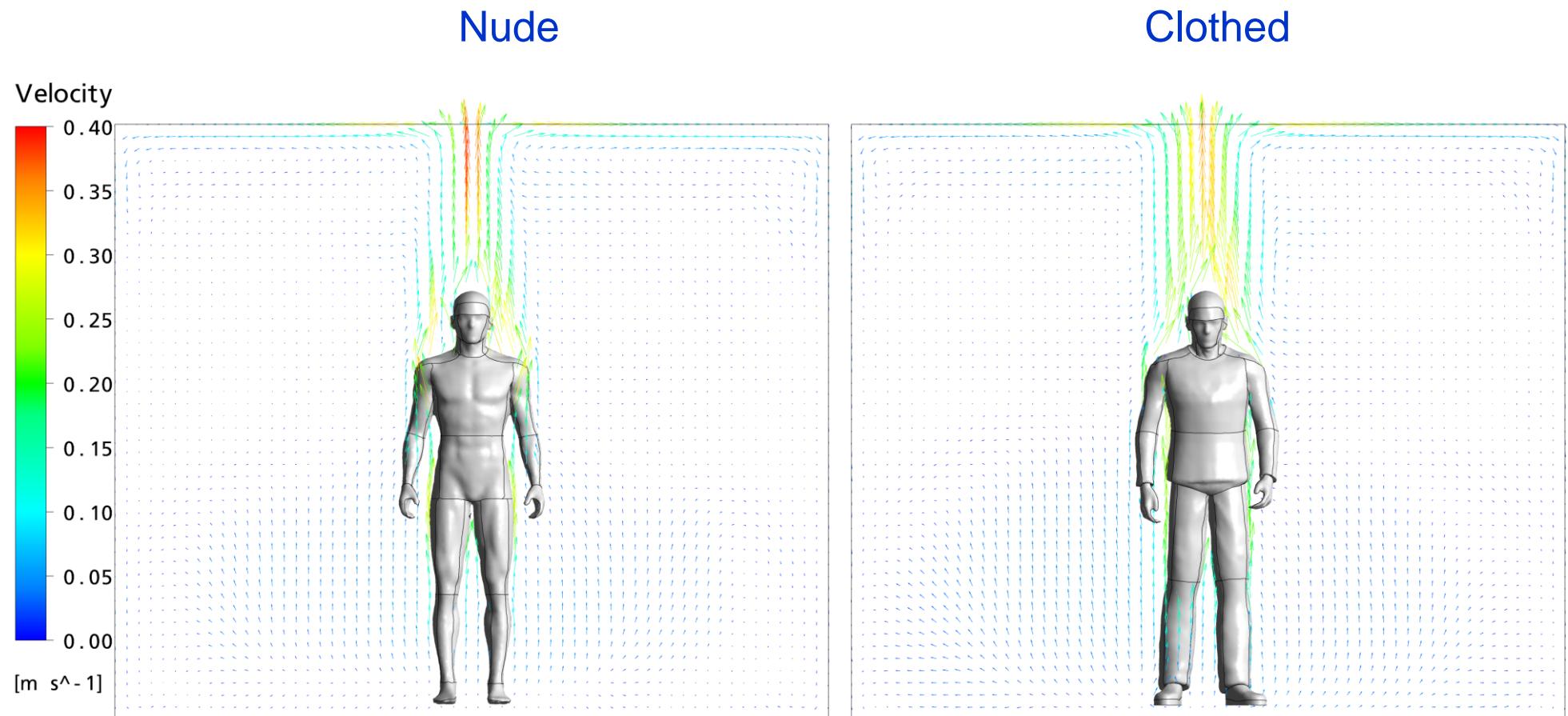
Temperature



Air speed



Velocity



Future Work

- Continued Validation
- Asymmetric radiation
- Ceiling radiant cooling
- Breathing manikins and IAQ
- Moving meshes

Acknowledgements

- EPSRC: Grant Ref. EP/C517520/2
- Mr Chris Staples and Dr Yehuda Sinai (Ansys UK)